

III. Laminar Solutions of N-S eqn.

- ② Consider Incomp flow, w/ const props & small ΔT
 5 eqns, 5 unknowns, u, v, w, p, T

Conti $\nabla \cdot \vec{V} = 0$ (1 eqn)

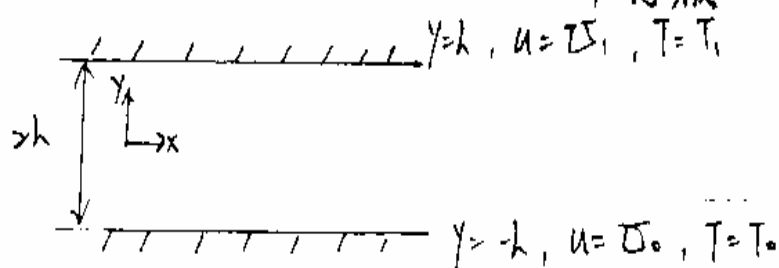
mom $\rho \frac{D\vec{V}}{Dt} = \rho \vec{g} - \nabla p + \mu \nabla^2 \vec{V}$ (3 eqn)

Energy $\rho C_p \frac{DT}{Dt} = k \nabla^2 T + \Phi$ (1 eqn)

energy eqn is uncoupled from conti & momentum eqn.

Example of Exact Laminar Solutions

- ② Flow between infinite parallel plates } parallel
 平行板



Assumptions

- (1) Incompressible (small ΔT)
 const properties
- (2) 2-D flow (no flow into or out of x-y plane)
 $(w=0, \frac{\partial u}{\partial z} = \frac{\partial v}{\partial z} = \frac{\partial T}{\partial z} = 0)$
- (3) // flow, ($v=0$)
- (4) gravity neglect in x-y plane

Eqns of Motion

Conti $\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = 0$ $(\nabla \cdot \vec{v}) = 0$

(3) (2)

$\Rightarrow \frac{\partial u}{\partial x} = 0$ u is not a function of x & z
 $u = u(y, t)$

x-mom:

$\rho \left\{ \frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z} \right\} = \rho g_x - \frac{\partial p}{\partial x} + \mu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2} \right)$

(3) (2) (4) Conti (2)

$\Rightarrow \rho \frac{\partial u}{\partial t} = - \frac{\partial p}{\partial x} + \mu \frac{\partial^2 u}{\partial x^2}$

y-mom: $\rho \left\{ \frac{\partial v}{\partial t} + u \frac{\partial v}{\partial x} + \dots \right\} = \rho g_y - \frac{\partial p}{\partial y} + \mu (\nabla^2 v)$

(3) (4) (3)

$\Rightarrow \frac{\partial p}{\partial y} = 0$ p is not a function of y .

z-mom: $\rho \left[\dots \right] = \rho g_z - \frac{\partial p}{\partial z} + \mu (\nabla^2 w)$

(2) (2)

$\frac{\partial p}{\partial z} = \rho g_z$, $\vec{g} = -g \vec{k}$, $\frac{\partial p}{\partial z} = -\rho g$

$p = -\rho g z + f(x, t)$

hydrostatic pressure

define

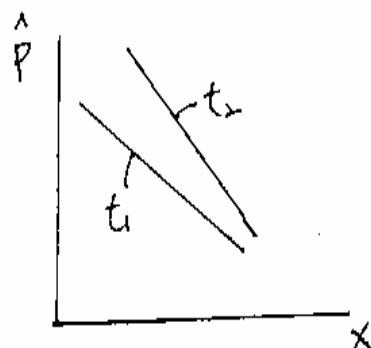
$\hat{p} = p + \rho g z$, $\hat{p} = f(x, t)$

So

$$\frac{\partial}{\partial x} \left\{ p \frac{\partial u}{\partial t} = - \frac{\partial \hat{p}}{\partial x} + \mu \frac{\partial^2 u}{\partial y^2} \right\}$$

$$\cancel{p \frac{\partial^2 u}{\partial x \partial t}} = - \frac{\partial}{\partial x} \left(\frac{\partial \hat{p}}{\partial x} \right) + \mu \cancel{\frac{\partial^3 u}{\partial x \partial y^2}} \rightarrow \left(\frac{\partial u}{\partial x} \right)$$

$\Rightarrow \frac{\partial}{\partial x} \left(\frac{\partial \hat{p}}{\partial x} \right) = 0$ $\hat{p}$ is a linear function of x .



Energy eqn.

$T = T_1$ entire plate $\Rightarrow T \neq T(x)$

$$\rho C_p \left[\frac{\partial T}{\partial t} + \cancel{u \frac{\partial T}{\partial x}} + \cancel{v \frac{\partial T}{\partial y}} + \cancel{w \frac{\partial T}{\partial z}} \right] = K \left(\cancel{\frac{\partial^2 T}{\partial x^2}} + \frac{\partial^2 T}{\partial y^2} + \cancel{\frac{\partial^2 T}{\partial z^2}} \right) + \dot{q}$$

(3) (2) (1)

$$\dot{q} = \underbrace{\mu \left[\frac{\partial u}{\partial y} \right]^2}_{\text{viscous dissipation}} + \lambda (\nabla^2 T)$$

$\rightarrow$ 速度的微分

$$\Rightarrow \rho C_p \frac{\partial T}{\partial t} = K \frac{\partial^2 T}{\partial y^2} + \mu \left(\frac{\partial u}{\partial y} \right)^2$$



非线性

$$\frac{\partial T}{\partial y} = 0$$

$$T = (y/2)^2$$

线性 $\frac{\partial T}{\partial y} = 0$

到板中心

热传导源

Steady flow sol.

$$\left. \begin{array}{l} u = u(y) \text{ only} \\ p = p(x) \text{ only} \\ T = T(y) \text{ only} \end{array} \right\} \frac{dp}{dx} = \text{Const}$$

Integrate

$$x\text{-mom} : \frac{d^2 u}{dy^2} = \frac{1}{\mu} \frac{dp}{dx} \rightarrow \text{O.D.E}$$

$$\text{Energy} : 0 = k \frac{dT}{dy^2} + u \left(\frac{du}{dy} \right)^2$$

Assume B.C. $U_1 = U$, $y=h$ for all time
 $U_2 = 0$, $y=-h$

$$\frac{du}{dy} = \frac{1}{\mu} \frac{dp}{dx} y + C_1$$

$$\xrightarrow{\text{Int}} u = \frac{1}{2\mu} \left(\frac{dp}{dx} \right) y^2 + C_1 y + C_2$$

Use B.C.s to find constants

$$\text{at } y=-h, u=0 \rightarrow 0 = \frac{h^2}{2\mu} \left(\frac{dp}{dx} \right) - C_1 h + C_2$$

$$y=h, u=U_1 \rightarrow U = \frac{h^2}{2\mu} \left(\frac{dp}{dx} \right) + C_1 h + C_2$$

$$C_1 = \frac{U}{2h}, \quad C_2 = \frac{U}{2} - \frac{h^2}{2\mu} \left(\frac{dp}{dx} \right)$$

$$u(y) = \frac{y^2 - h^2}{2\mu} \left(\frac{dp}{dx} \right) + \frac{U}{2h} y + \frac{U}{2}$$

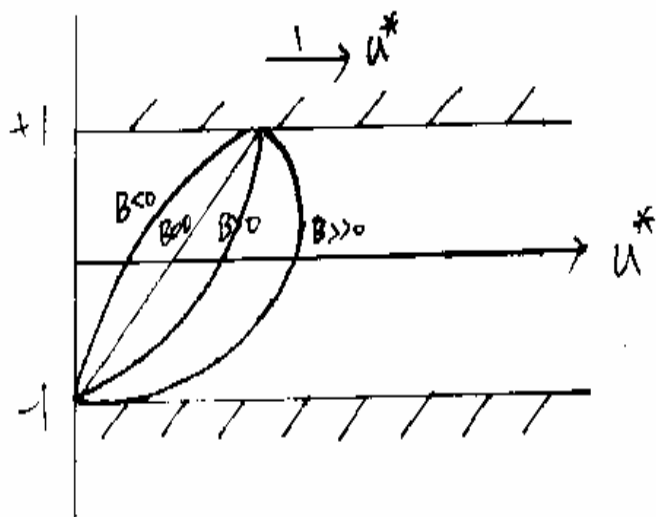
Non-dimensional form

$$u^* = \frac{u}{U}, \quad y^* = \frac{y}{h}$$

$$B = \text{pressure gradient parameter} = -\frac{h^2}{4U} \left(\frac{dp}{dx} \right)$$

$$u^* = \frac{(1-y^{*2})}{2} B + \frac{y^{*2} + 1}{2}$$

General steady flow sol. for flow between 2 infinite // plates.



$$y^* \rightarrow 0 \rightarrow u^* = \frac{B}{2} + \frac{1}{2}$$

$$B > 0, \quad u^* = \frac{1}{2}$$

$$B > 0, \quad u^* > \frac{1}{2}$$

$$B < 0, \quad u^* < \frac{1}{2}$$

$$* B >> 0, \quad u^* \text{ will } > 1$$

在不可壓縮流場

中, ρ 是 const 所以

溫度可由速度來推。

Energy Equation.

$$k \frac{d^2 T}{dy^2} = -u \left(\frac{du}{dy} \right)^2$$

non-dimensionalize

$$\begin{cases} y^* = y/h \\ u^* = u/\tau_s \end{cases}$$

define

$$T^* = \frac{T - T_0}{T_1 - T_0}$$

at

$$y=h, T=T_1, T^*=1$$

$$y=0, T=T_0, T^*=0$$

$$\frac{dT}{dy} = \frac{dT}{dT^*} \frac{dT^*}{dy^*} \frac{dy^*}{dy}, \quad T = T_0 + T^*(T_1 - T_0)$$

$$\frac{d[T_0 + T^*(T_1 - T_0)]}{dT^*} = (T_1 - T_0) \quad \rightarrow \quad \frac{dy^*}{dy} = \frac{d(y/h)}{dy} = \frac{1}{h}$$

$$\frac{dT}{dy} = \frac{dT}{dT^*} \frac{dT^*}{dy^*} \frac{dy^*}{dy} = (T_1 - T_0) \left(\frac{dT^*}{dy^*} \right) \frac{1}{h}$$

$$\frac{d^2 T}{dy^2} = \frac{d}{dy^*} \frac{dy^*}{dy} \left(\frac{dT}{dy} \right) = \frac{T_1 - T_0}{h^2} \frac{d^2 T^*}{dy^{*2}}$$

$$\frac{du}{dy} = \frac{\tau_s}{h} \frac{du^*}{dy^*}$$

$$\frac{d^2 T^*}{dy^{*2}} = \frac{-u}{K} \frac{h^2}{(T_1 - T_0)} \frac{\tau_s^2}{h^2} \left(\frac{du^*}{dy^*} \right)^2$$

$$\text{原 } K \frac{d^2 T}{dy^2} = -u \left(\frac{du}{dy} \right)^2$$

$$\frac{du^*}{dy^*} = \frac{1}{2} - B y^*$$

Let $Br = \text{Brinkman \#} = \frac{\mu U^2}{k(T_i - T_o)}$

$$\frac{d^2 T^*}{dy^{*2}} = -Br \left(\frac{1}{4} - By^* + B^2 y^{*2} \right)$$

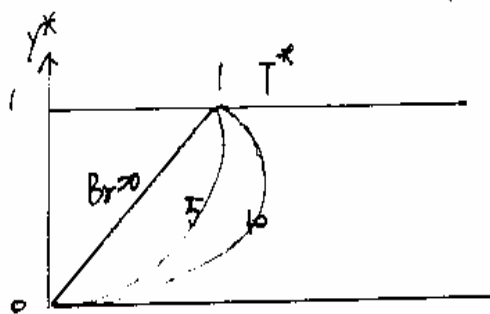
integrate twice:

$$T^* = \frac{1}{2} (1+y^*) + \frac{Br}{8} (1-y^*)^2 - \frac{BBr}{6} (y^* - y^{*3}) + \frac{B^2 Br}{12} (1-y^{*4})$$

case a). $B=0$ (No pressure gradient $\rightarrow$ pure Couette flow) /kwet/

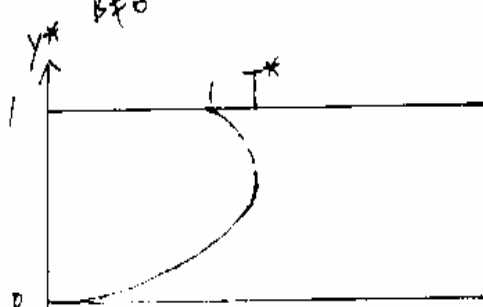
only valid for small ΔT

$\rightarrow$ assume constant properties



$Br=0 \rightarrow U=0$,
no flow fluid --
conduction.

case b). $B \neq 0$



H.W. -2

prob 2-18

prob 3-8

air at 20°C

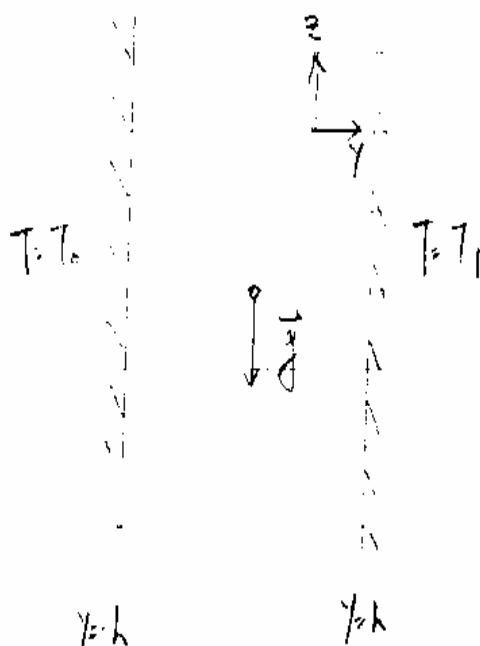
$$\mu = 1.81 \times 10^{-4} \text{ Pa}\cdot\text{s}$$

1.2 Non-Newtonian Couette Flow

e.g. Suppose

$$\tau_{ij} = -p \delta_{ij} + \underbrace{\eta}_{\text{const}} \left(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right)$$

> Free Convection between Vertical plates



Assume:

- 1) zero-pressure gradient
- 2) Infinite, stationary walls
- 3) Steady flow ($\frac{\partial}{\partial t} = 0$)
- 4) 2-D ($u = \frac{\partial}{\partial x} = 0$)
- 5) // flow ($v = 0$)
- 6) ρ is a weak function of T (ΔT is small)
(slightly compressible) ($\rho \approx \rho_0 (1 - \beta \Delta T)$)
- 7) μ & k are constants

If without flow $\frac{\partial p}{\partial z} = -\rho g$

with flow

$$\rho \frac{Dw}{Dt} = -\frac{\partial p}{\partial z} - \rho g + \mu \nabla^2 w$$

In general $\rho = \rho(p, T)$

$$d\rho = \left(\frac{\partial \rho}{\partial p} \right)_T dp + \left(\frac{\partial \rho}{\partial T} \right)_p dT = A'$$

Assume $\rho =$ function of T but not p

$$\begin{aligned} \beta &\equiv \text{coefficient of thermal expansion} \\ &\equiv -\frac{1}{\rho} \left(\frac{\partial \rho}{\partial T} \right)_p \end{aligned}$$

$$dp = -\rho \beta dT$$

$$= -\rho_0 \beta dT$$

let $\rho = \rho_0 + d\rho$

$$\rho = \rho_0 - \rho_0 \beta (T - T_0)$$

"slightly compressible flow assumption"

@ assume $\rho = \rho_0 = \text{constant}$ for all terms, except the gravity term.

z-mom

$$\rho_0 \frac{DW}{Dt} = -\frac{\partial p}{\partial z} - \rho_0 g + \underbrace{\rho_0 \beta (T - T_0) g}_{\text{Buoyancy term} \Rightarrow \text{力项}} + \mu \nabla^2 W$$

let $\hat{p} = p + \rho_0 g z$

= hydrodynamic pressure + total hydrostatic pressure
 $\nearrow$ 力 $\nearrow$ 力

Zero pressure gradient

$$\rightarrow \frac{\partial p}{\partial z} \approx 0$$

z-mom

$$\rho_0 \frac{DW}{Dt} = -\frac{\partial \hat{p}}{\partial z} + \rho_0 \beta (T - T_0) g + \mu \nabla^2 W$$

Conti

$$\frac{\partial \hat{p}}{\partial t} + \frac{\partial \hat{p}}{\partial x} u + \frac{\partial \hat{p}}{\partial y} v + \frac{\partial \hat{p}}{\partial z} w \approx 0$$

3) 4) 5)

ρ_0

$$\rho \frac{\partial W}{\partial z} + W \frac{\partial \rho}{\partial z} \approx 0$$

Since wall temp is constant, $T = f(z)$

↓

$$\frac{\partial W}{\partial z} \approx 0, W = W(y) \text{ only}$$

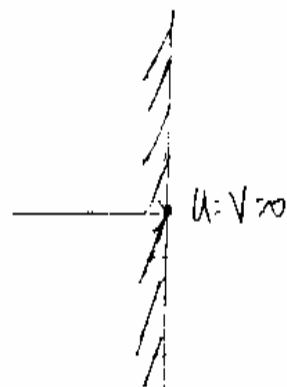
$$\rightarrow T = T(y) \text{ only}$$

$$\rightarrow \rho = \rho(y) \text{ only}$$

y-mom

$$\rho \frac{DV}{Dt} = -\frac{\partial p}{\partial y} + \rho g + \mu \nabla^2 V$$

5) z-dir only 5)



$$\Rightarrow \frac{\partial p}{\partial y} \approx 0, \quad \frac{\partial p}{\partial x} \approx 0, \quad \frac{\partial p}{\partial t} \approx 0$$

$$p = p(z) \text{ only, but } \hat{p} = \hat{p}(z)$$

$$\hat{p} = \text{const everywhere (free convection flow is not driven by pressure, but by temperature differences)}$$

z-mom

$$\rho_0 \left(\frac{\partial w}{\partial t} + u \frac{\partial w}{\partial x} + v \frac{\partial w}{\partial y} + w \frac{\partial w}{\partial z} \right) = -\frac{\partial p}{\partial z} + \rho_0 \beta (T - T_0) g + \mu \left(\frac{\partial^2 w}{\partial x^2} + \frac{\partial^2 w}{\partial y^2} + \frac{\partial^2 w}{\partial z^2} \right)$$

3) 4) 5) conti 1) 4) conti

$$\Rightarrow \mu \frac{dw}{dy^2} = -\rho_0 \beta g (T - T_0)$$

Energy

$$\rho_0 C_p \frac{DT}{Dt} = K \nabla^2 T + \Phi$$

$$0 = K \frac{\partial^2 T}{\partial y^2} + \mu \left(\frac{dw}{dy} \right)^2$$

$$Gr = \text{Grashof \#} = \frac{g \rho^2 h^3 \beta (T - T_m)}{\mu^2}$$

$$J = \text{nameless} = \frac{\beta g \mu h}{k}, \quad J \ll 1 \text{ for all fluids}$$

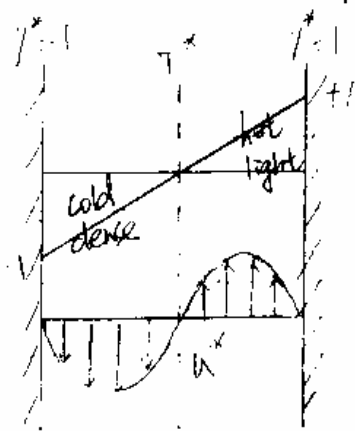
$$\frac{d^2 T^*}{dy^{*2}} = - \frac{J}{Gr} \left(\frac{dw^*}{dy^*} \right)^2 \quad J \ll Gr$$

$$\text{So } \frac{d^2 T^*}{dy^{*2}} = 0 \quad \rightarrow \text{sl'n } T^* = y^*$$

$$\text{mom eqn. } \frac{dw^*}{dy^{*2}} = -Gr y^*$$

$$\text{B.C. } w^* = 0, \quad @ y^* = \pm 1$$

$$w^* = \frac{Gr}{6} (y^* - y^{*3})$$



define 寬長比為10倍
為二流表板

3. Unsteady // flow

3.1 Rayleigh Problem Impulse start of an infinite flat plate.

